

Repair and Retrofit of a Coke Drum Skirt Attachment Weld

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Problem Overview

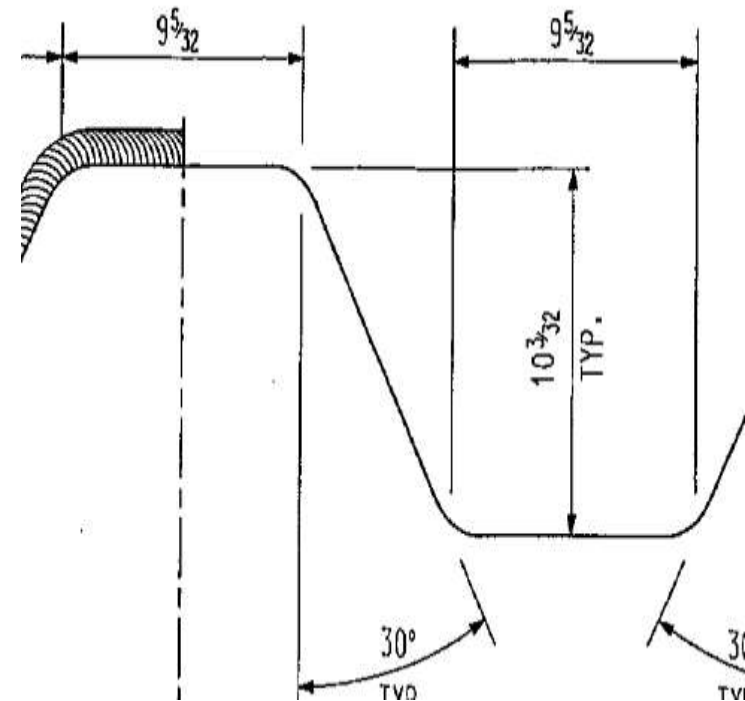
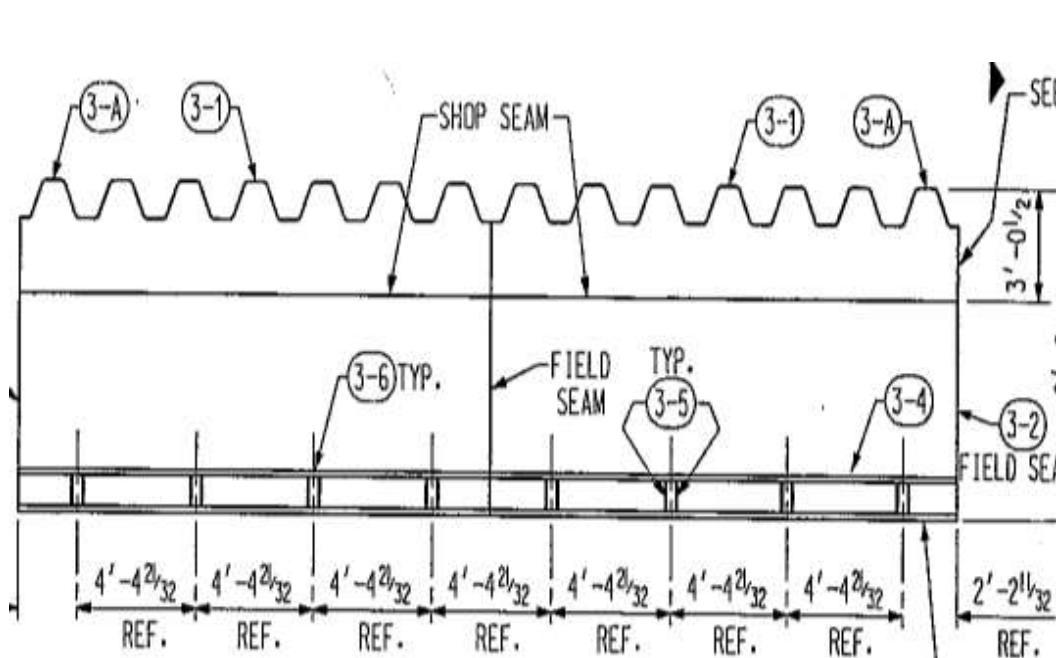
- Extensive cracking noted on skirt attachment weld of relatively young coke drums
- Needed to find a method to repair and extend the life of the skirt attachment to maximize the life of drums otherwise in very good condition
- Very short time frame to TA to analyze, design, fabricate, and implement solution (June-Sep)

Presentation Overview

- Background
- Inspection of the cracked attachment welds
- Fitness-for-service assessment
- Retrofit plan
- Implementation of retrofit
- Subsequent re-inspection results

Background

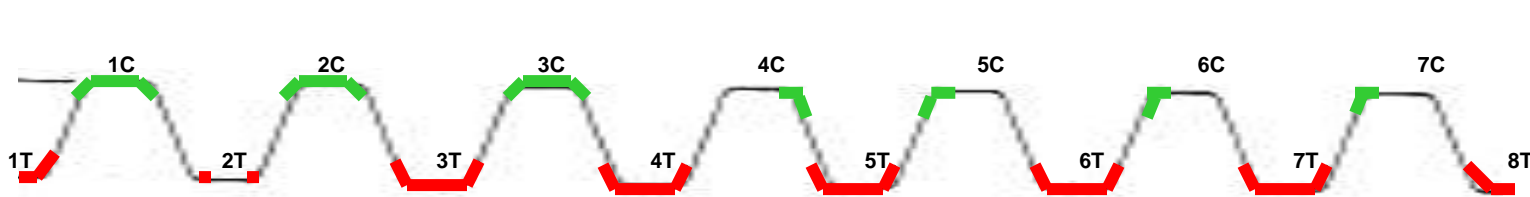
- 1-1/4 Cr – 1/2 Mo drums installed in 2007
- Drums Dimensions: 22' – 0" ID x 72' – 4" TT
- The skirt-to-shell attachment incorporates a scallop type lap-joint design with a continuous full leg fillet weld



Background & Inspection

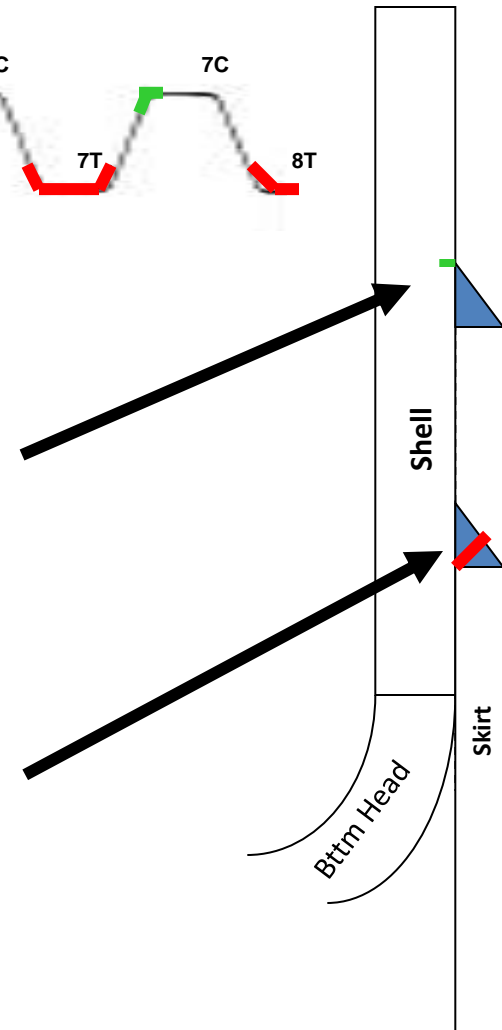
- Routine skirt attachment inspection scheduled prior to the Fall 2011 Turnaround
- Inspection was completed in June while the drums were in operation
- Extensive cracking noted during the inspection extending nearly full circumference of both drums
- Cracks were sized with dry, color contrast, MT and depths determined using UTPA
- The attachment weld was primarily cracked in two areas:
 - In the shell at the skirt attachment crown.
 - Through the throat of the trough fillet welds.

Skirt weld cracks



Peak cracks were at the toe of the fillet weld and in the base metal. Cracks were determined to be less than 0.10" deep.

Trough cracks were in the fillet weld throat. Cracks which did not extend through the full weld thickness appeared to originate at the weld root.

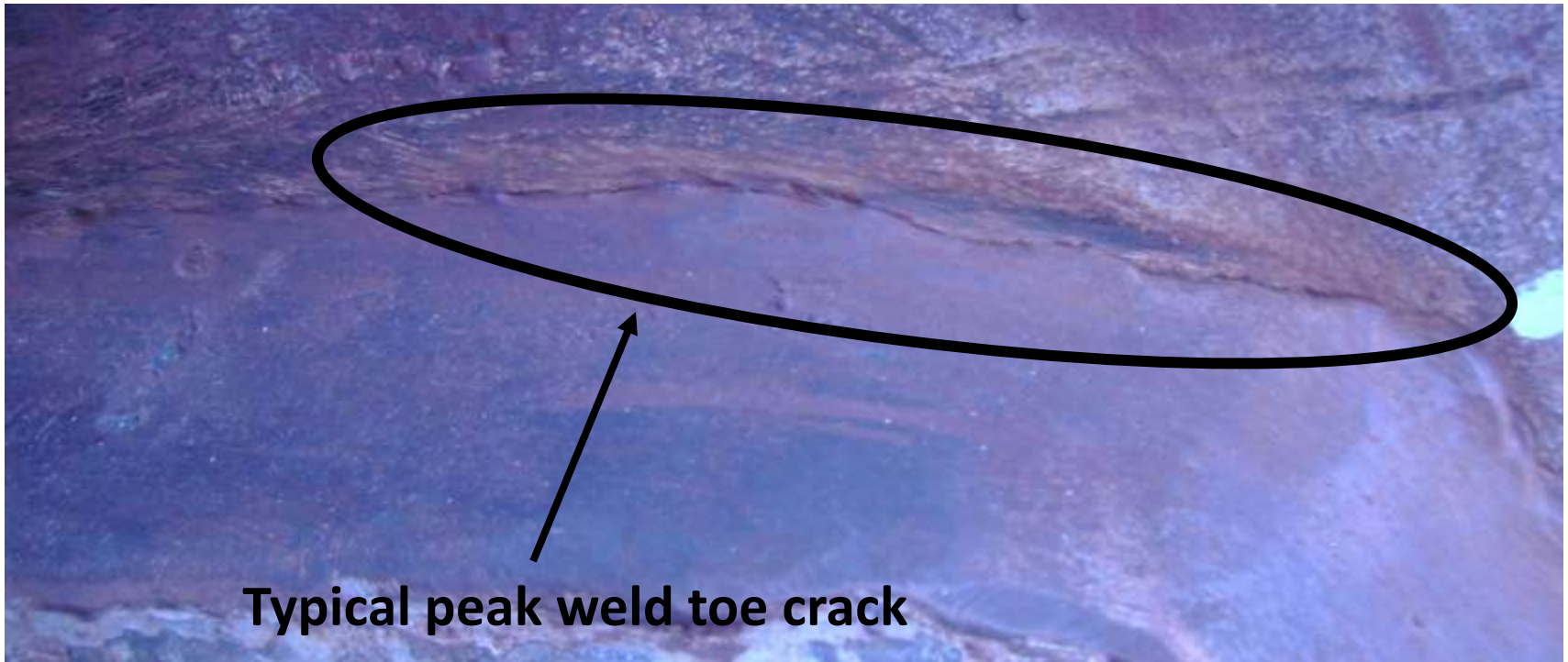


Trough Cracks



Typical trough fillet weld crack

Peak Cracks



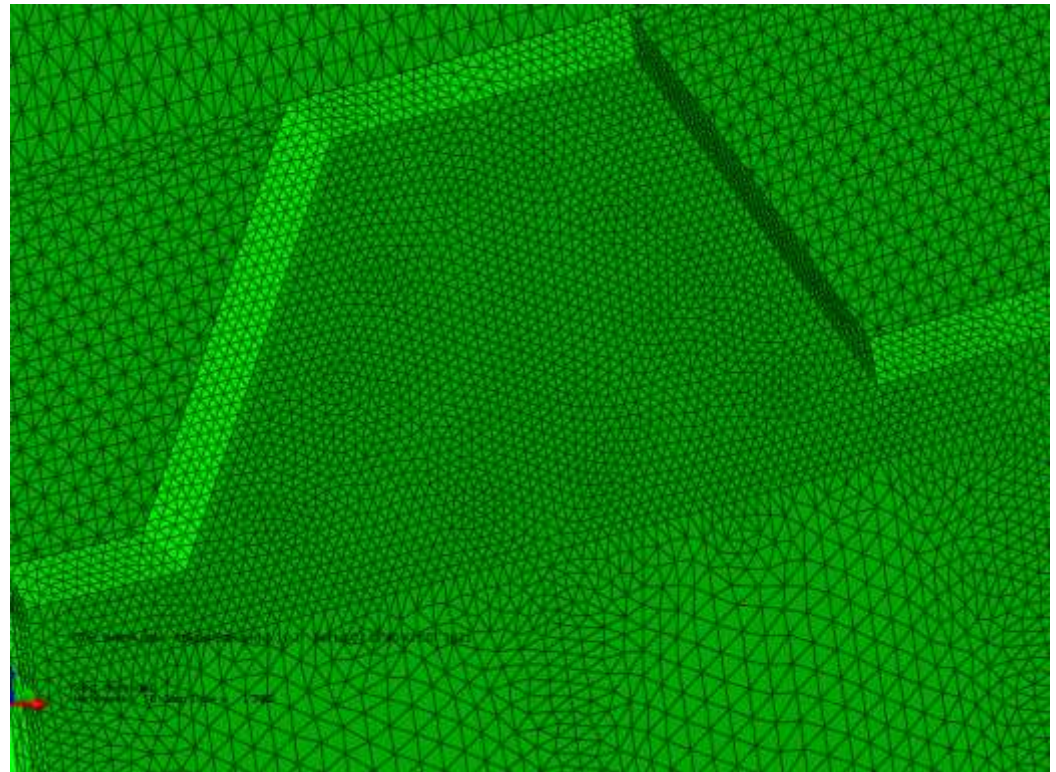
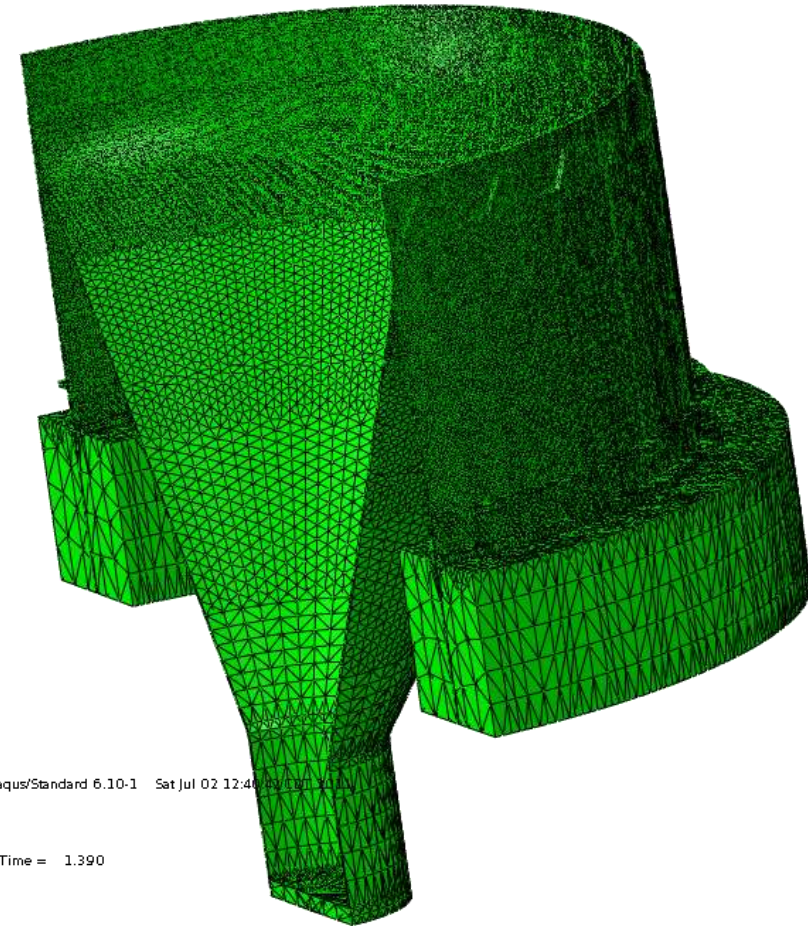
CHS' Approach to the Problem

- Four main issues
 - Fitness-for-Service evaluation for continued operations
 - Repair plan if necessary to allow continued operation until the scheduled turnaround
 - Long term repair approach needed
 - Short timeframe of ~10 weeks (mid June – Early September)
 - Develop & evaluate design alternatives
 - Detailed design
 - Material procurement and prefab to support installation during TA
- Procurement, fabrication and installation handled directly by CHS.

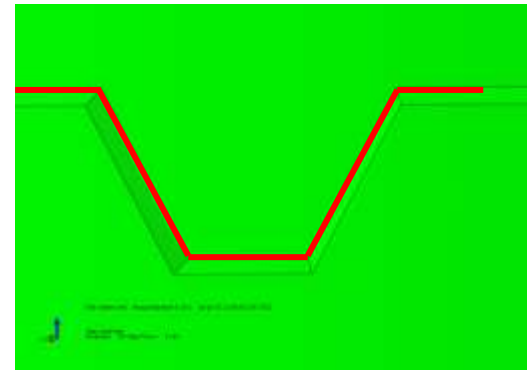
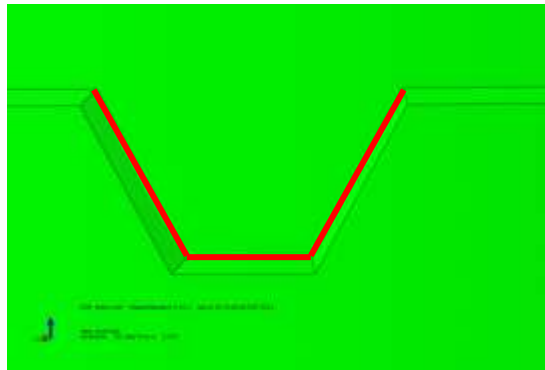
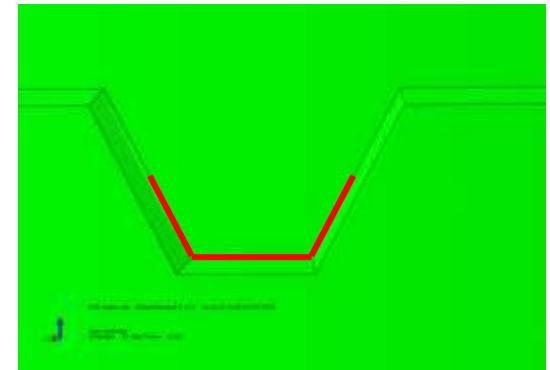
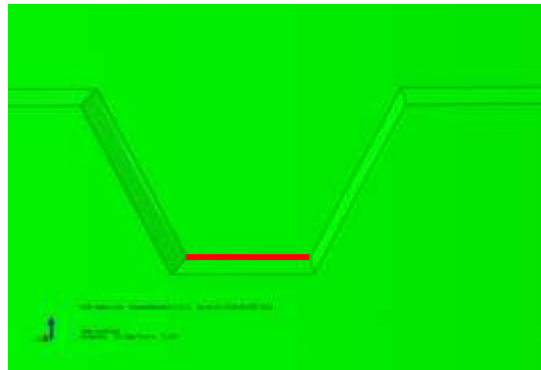
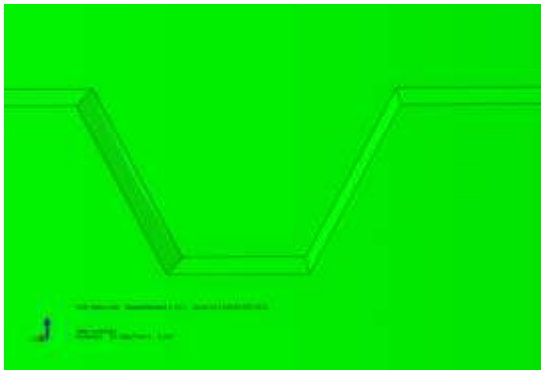
Fitness-For-Service Analysis of Skirt Stability

- Mechanical loads at operating and corroded condition:
 - Drum Weight
 - Seismic moment and shear force.
 - Wind moment and shear force.
- Thermal loads during typical filling and quenching temperature profiles.

Finite Element Model



Model Cases 1 to 7



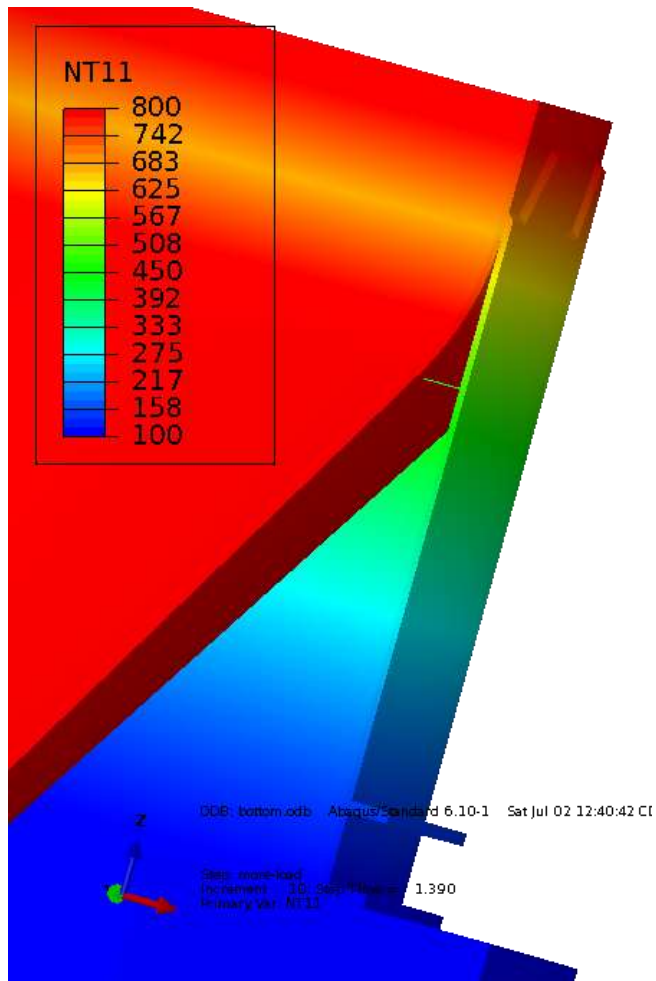
- Model Case 1 : Fill, ASME Load Combination 5 , No Weld Crack.
- Model Case 2 : Fill, ASME Load Combination 5 , Crack Size of 9.16 "
- Model Case 3 : Fill, ASME Load Combination 5 , Crack Size of 20.81 "
- Model Case 4 : Fill, ASME Load Combination 5 , Crack Size of 32.47 "
- Model Case 5 : Fill, ASME Load Combination 5 , Crack Size of 37.05 "
- Model Case 6 : Fill, ASME Load Combination 1, Crack Size of 37.05 "
- Model Case 7 : Quench, ASME Load Combination 1, Crack Size of 37.05 "

Load combinations of the 2007 edition of API 579/ ASME FFS:

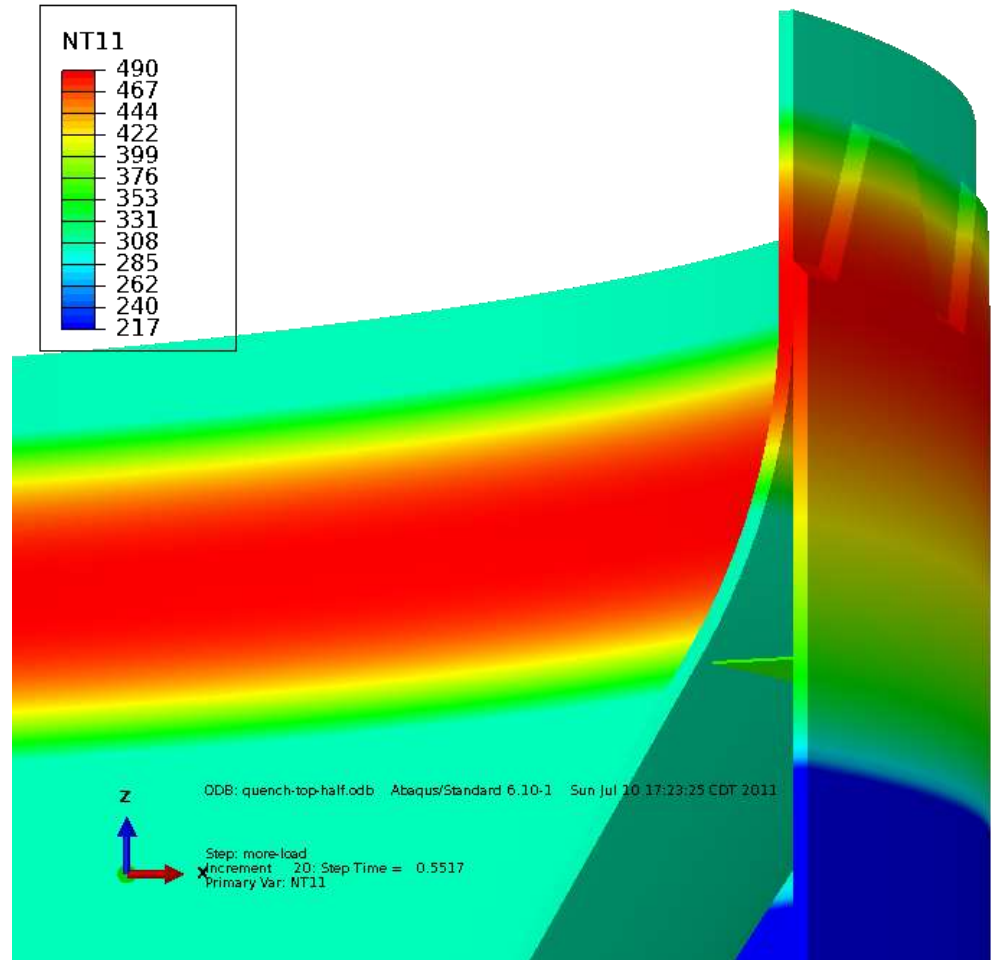
- Load combination 1: βD , where D is drum weight and β equals 2.7.
- Load combination 5: $0.86\beta D + 0.71\beta E$, where E is the seismic loads.

Thermal Loads

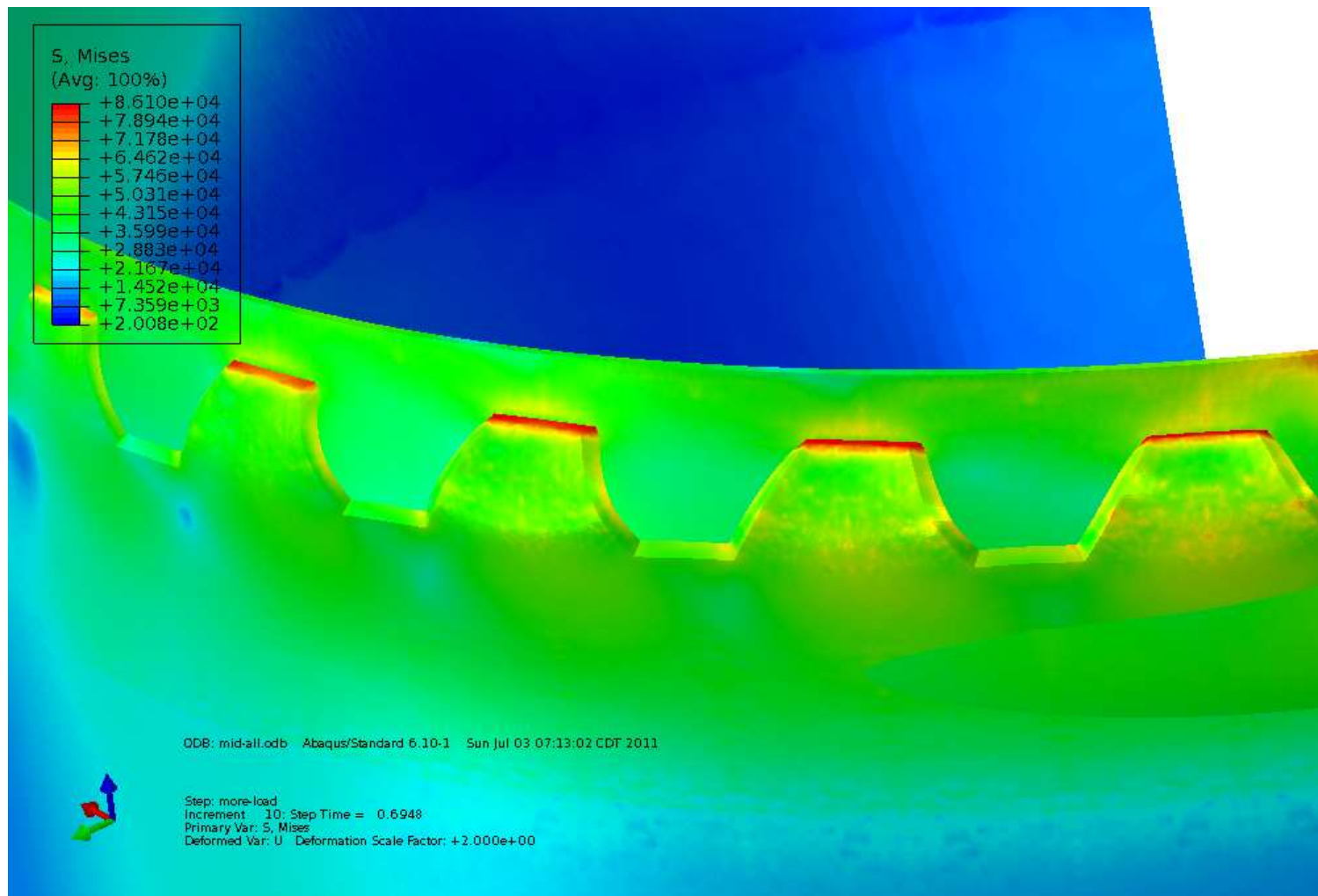
Fill



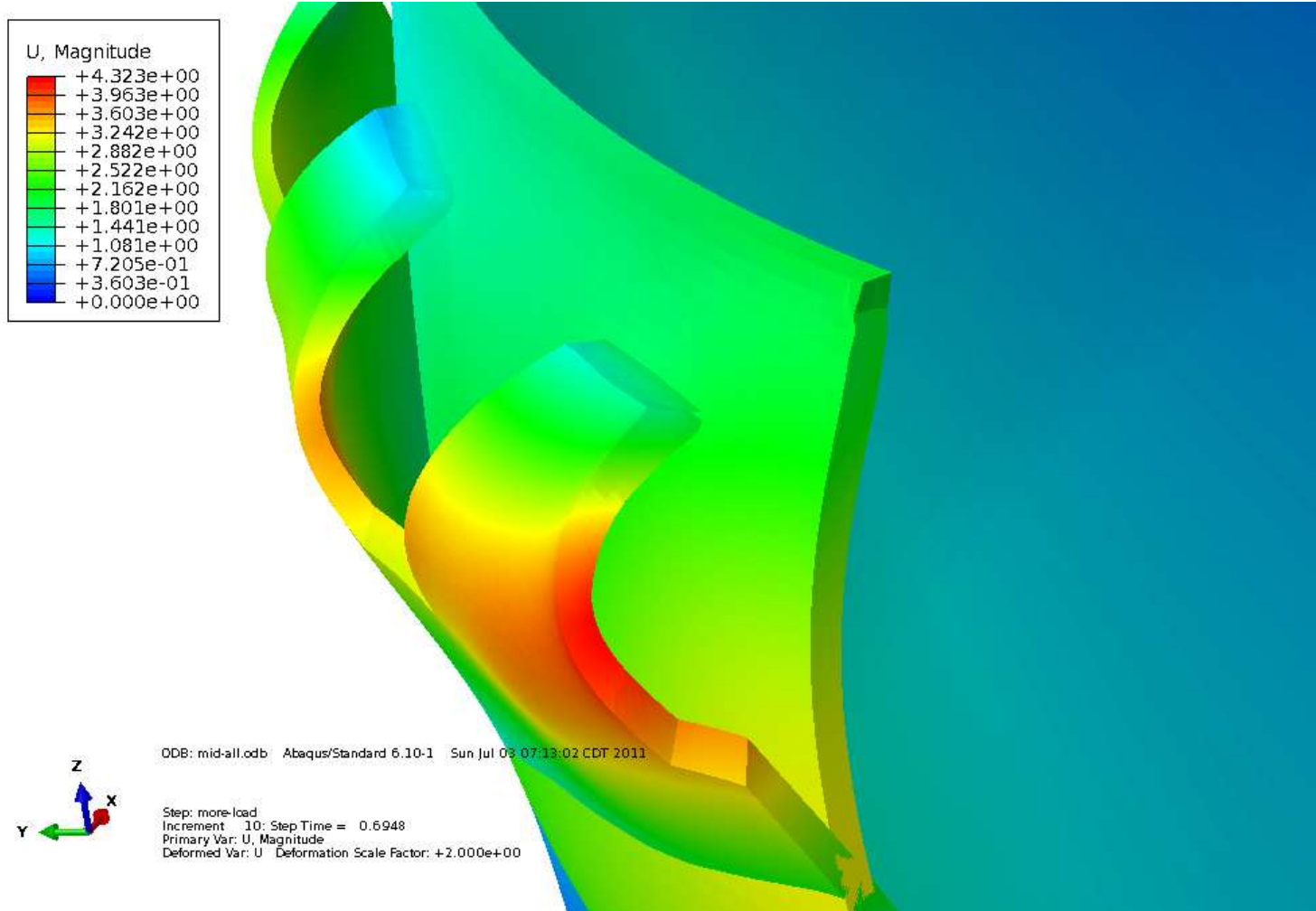
Quench



Von Mises Stress @ Collapse (Case 4)

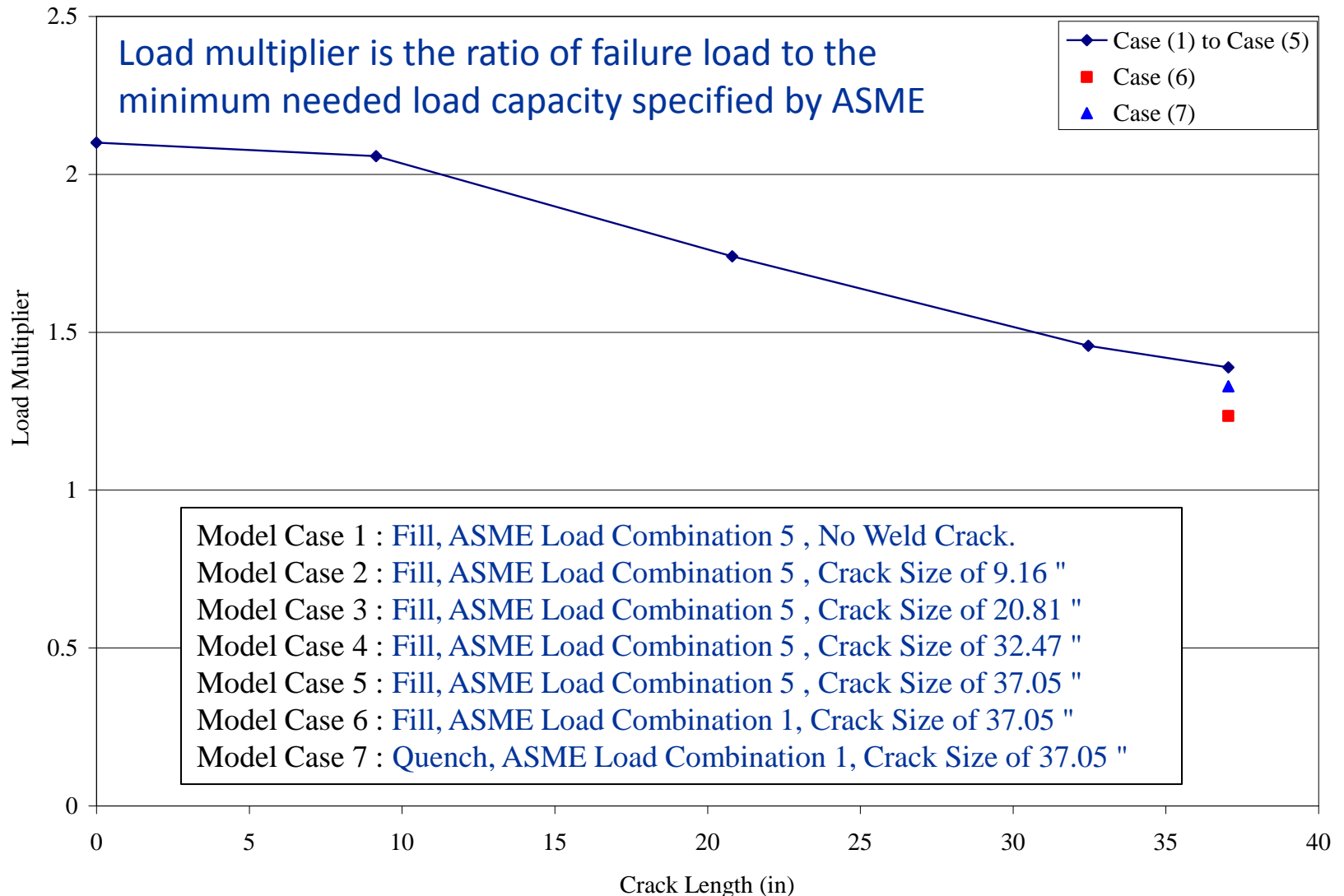


Displacement Magnitude @ Collapse (Case 4)



Summary of Results

Load Multiplier vs. Crack Length Under Seven Model Cases

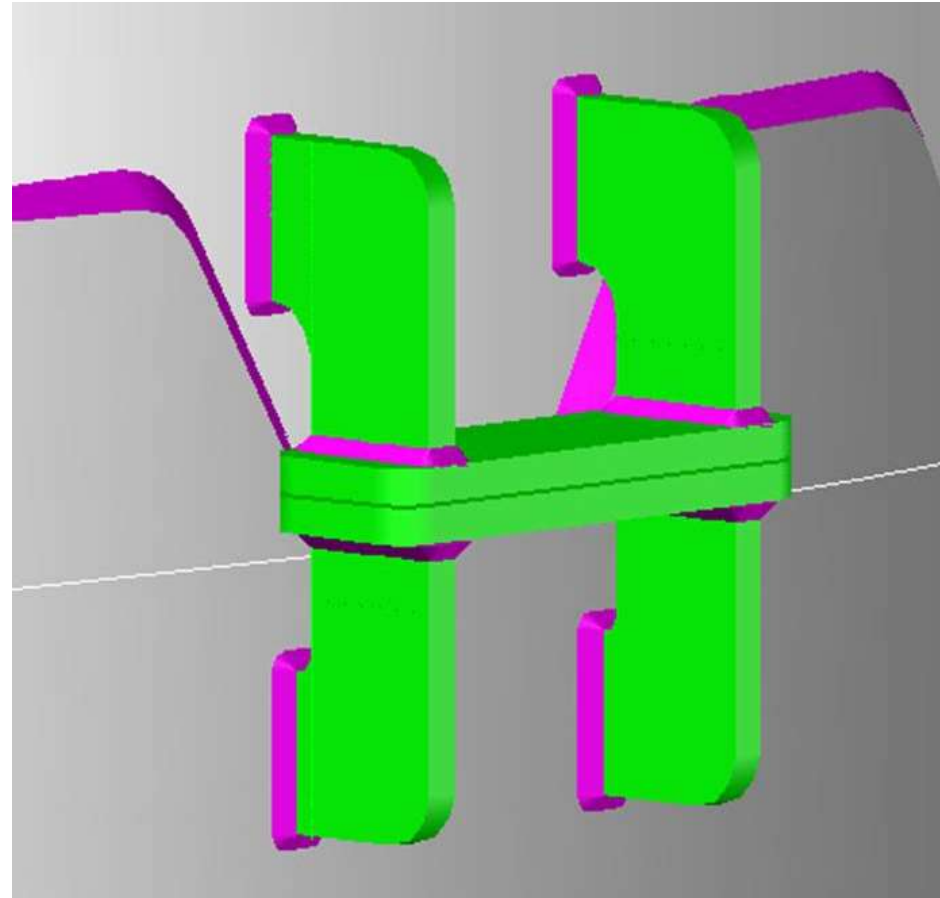


Summary of FFS Analysis

- For the longest crack of 37.05", the minimum load multiplier was 1.23. This means that the remaining ligament of the weld (half of the top weld attachment) had 23% more strength than that required by API-579/ ASME-FFS.
- The combination of drum weight and filling temperature profile was the worst load case.
- Skirt should remain stable with sufficient margin as long as one half of the peak of the scallop remains intact.
- Frequent inspections used to monitor crack propagation until TA.

Long Term Repair Approach

- Install sliding brackets for load transfer
- Allow through-throat cracks in scallop troughs to continue to propagate
- Manage crack propagation at scallop peaks by crack removal and addition of planned notches



28 brackets per drum

Retrofit Details

- Bracket fabricated from 1-1/4 Cr – 1/2 Mo
- Inconel 625 temper-bead welding
- Corners rounded and welds ground smooth.
- Matching sliding plates pre-loaded prior to welding.
- Nitronic 60 insert plates used to minimize galling.
- Brackets located to maximize access to scallop peaks for future repairs.

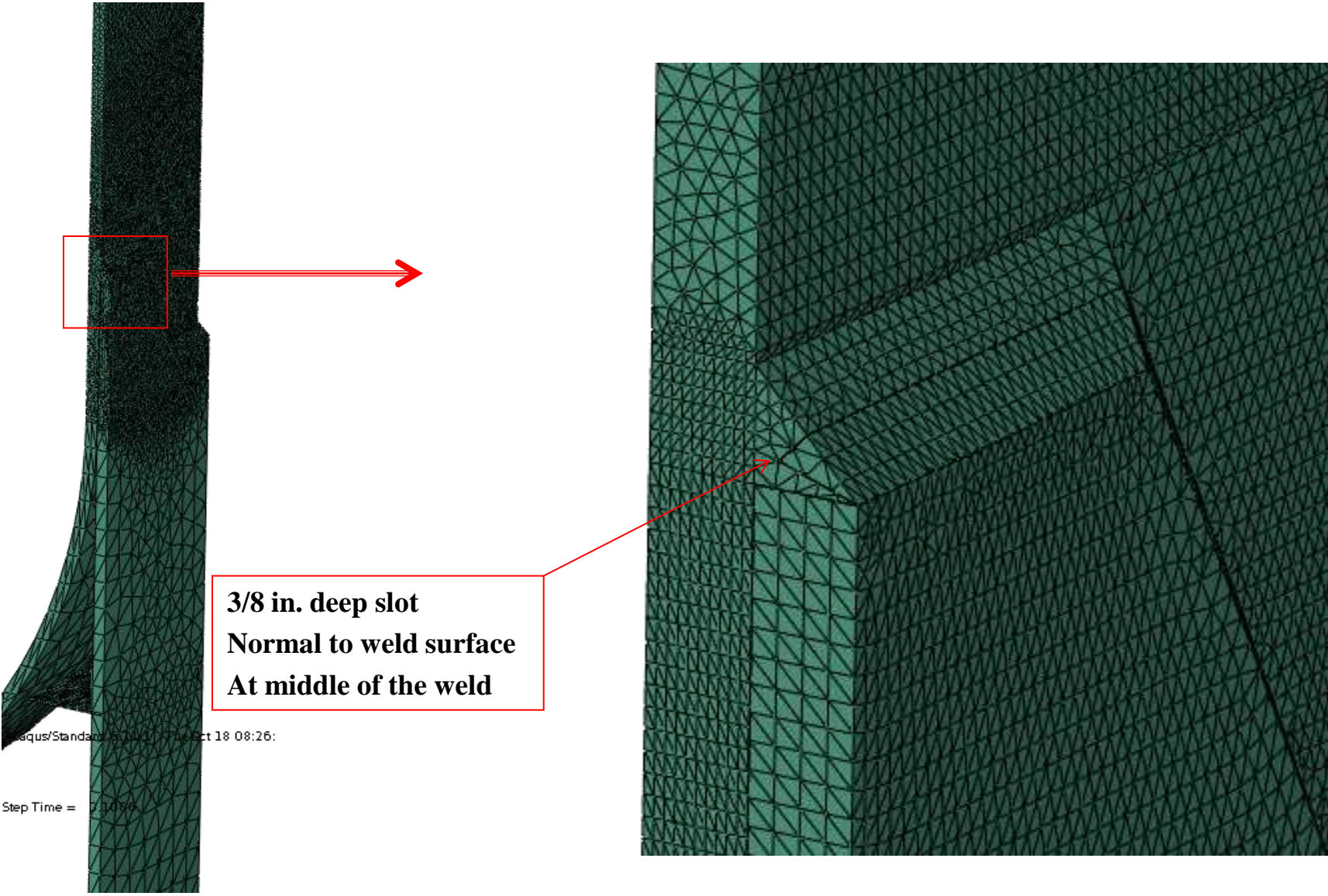
Summary of Retrofit Analysis

- Retrofit replacement model plastically collapsed at 3.667 times the operating weight of the drum.
- Using API-579/ ASME-FFS Standard of 2007 which requires a factor of 3.15, this bracket has adequate strength to resist nominal loads
- The repad (or filler plate) is an option that can potentially improve the logistics of field implementation. Its inclusion does not materially impact the safety margin of the retrofit which is controlled by the bending capacity of the bracket plate.

Analysis of Slotting Effect

- Examine the effectiveness of adding slots in “steering” cracks away from pressure boundary material & into fillet weld
- Approach
 - Simulate ~3/8” deep slots in the fillet attachment throat of scallop peaks
 - Consider mechanical and thermal transient loads
 - Exclude retrofit brackets (not expected to redistribute stresses until after significant cracking /unloading occur).
 - The bottom of the scallop weld is cracked half way up.
 - Determine impact of slots on fatigue life of drum side of fillet attachment per ASME 2004 elastic analysis.

Finite Element Model



**3/8 in. deep slot
Normal to weld surface
At middle of the weld**

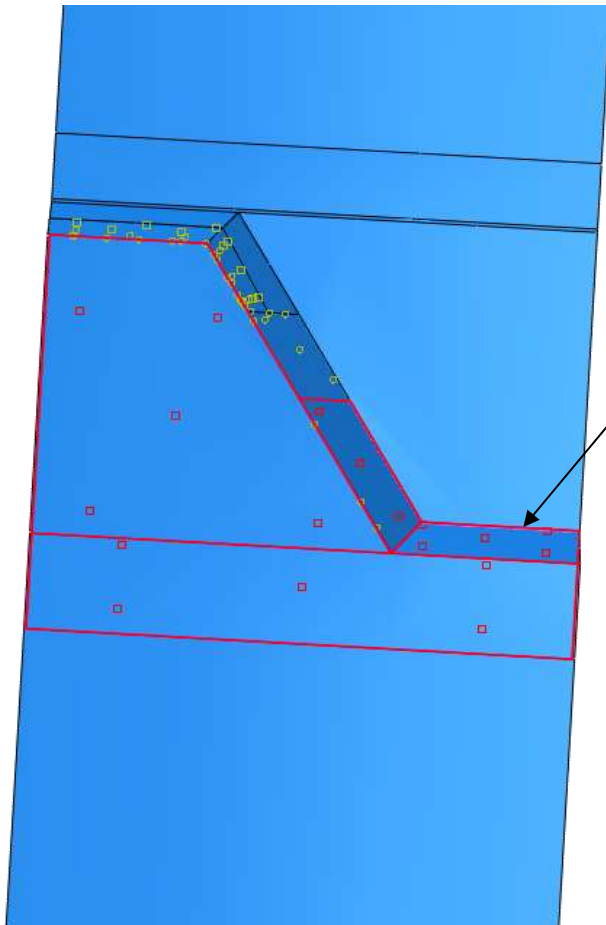
Step Time = 0.1096

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Two Configurations

Base Model

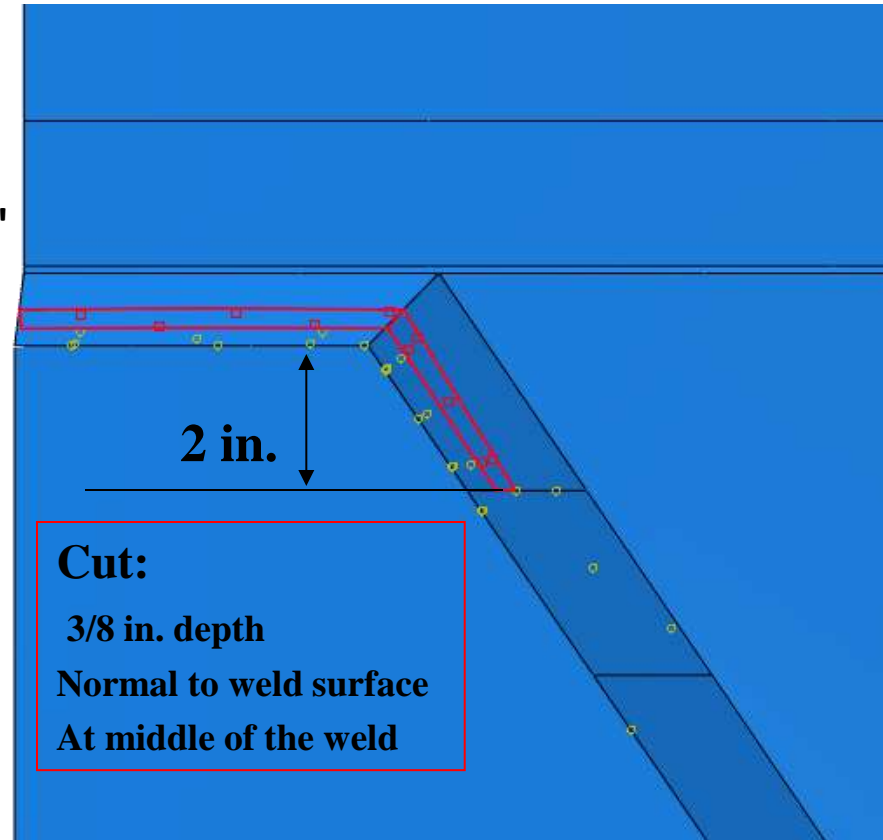
no slot



Crack Size of 10.4"

Slot Model

Same as Base Model + a slot cut across the peak of the scallop weld



2 in.

Cut:

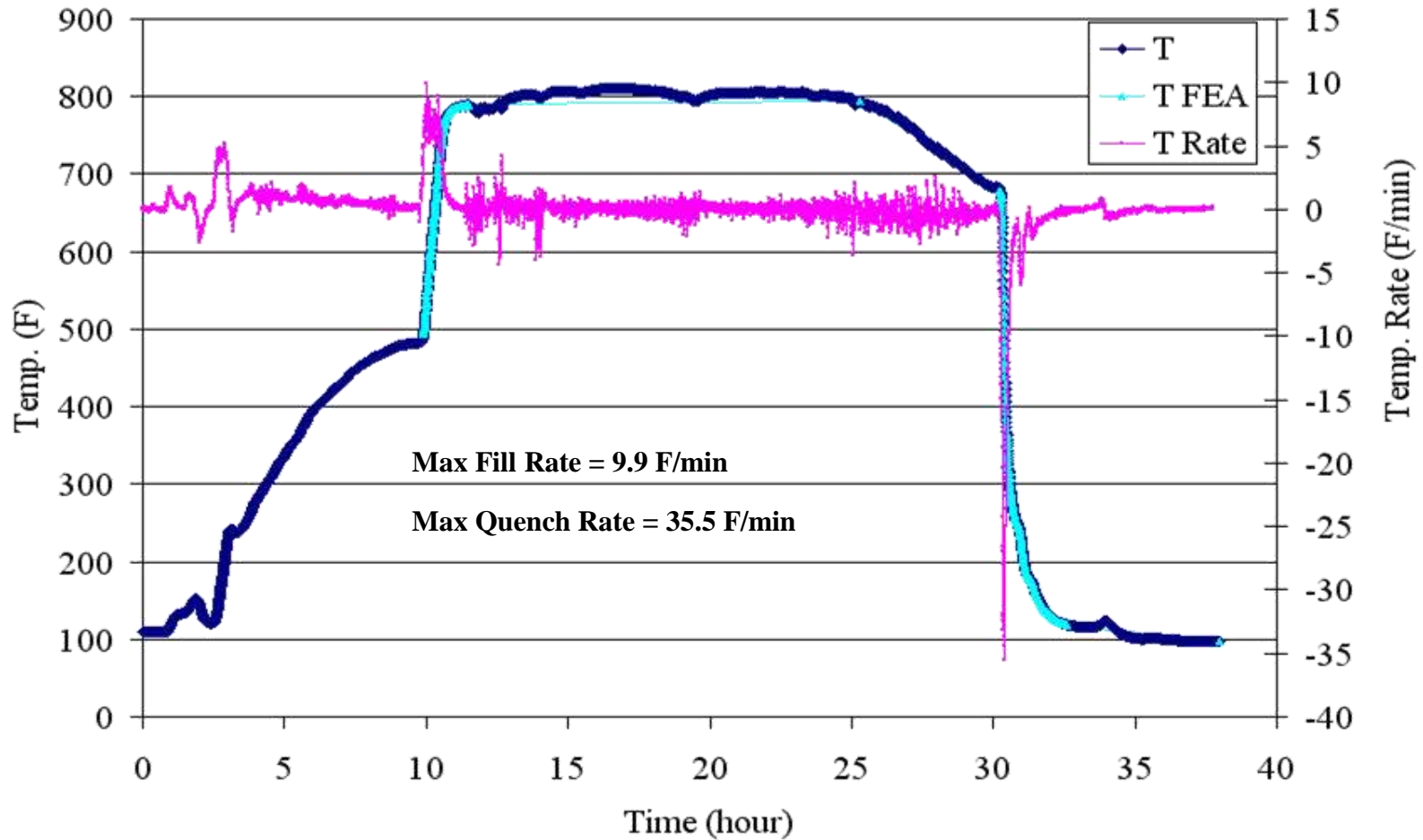
3/8 in. depth

Normal to weld surface

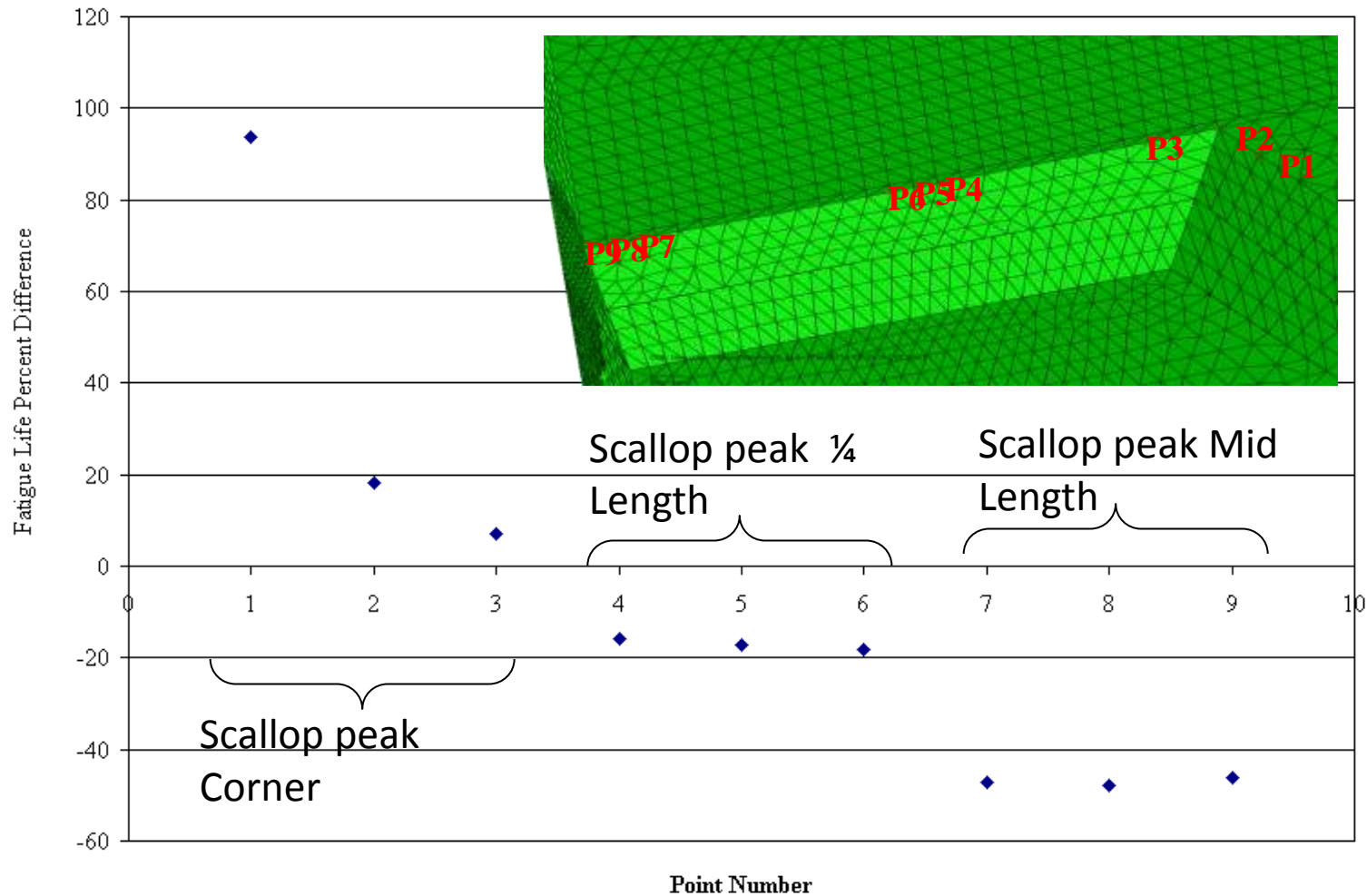
At middle of the weld

Representative Cycle

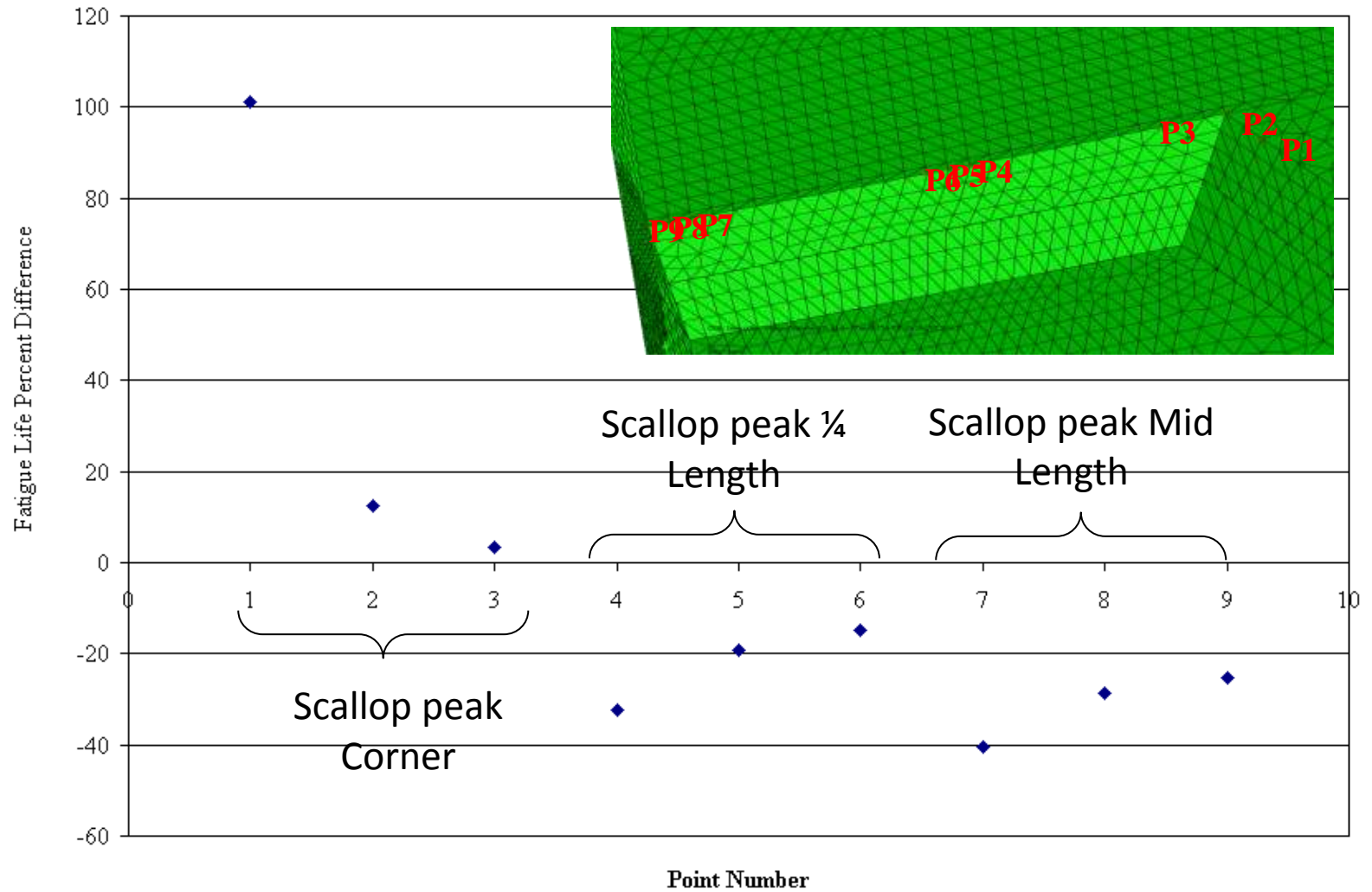
Measured Temp. vs. Temp. Used by FEA



Fatigue Life % Difference at Weld Toe



Fatigue Life % Difference next to Weld Toe



Summary of Slot Analysis

- The simulated slots cause fatigue life to increase at the scallop corner and decrease along the middle of the scallop peak. Therefore, fatigue life increases where it matters most (i.e. where life is shortest at the corner).
- For a life that is over 100,000 cycles which is an order of magnitude longer than the life of a typical drum, the peaks of cracked scallop welds in the skirt-to-drum attachment weld are expected to have a long fatigue life with and without specified slots.
- The slots are expected to direct a crack that starts at the root of the scallop peak weld to propagate along the throat and away from the pressure boundary.

Implementation of Retrofit

- Preheat maintained via heating elements inside the drum
- Planned 15 days mechanical duration met for installation
- Preload to sliding plates applied using a press and welding hold tabs in place
- Slots were added in half of scallop peaks
- Challenge implementing required weld quality
- Fabrication tolerance challenge

Subsequent Inspection

- A re-inspection of sliding brackets and skirt attachment weld cracking was completed in August 2012.
- Inspection Results
 - No cracking noted in attachment fillet weld connections using dry MT
 - Brackets had no distortion or notable movement
 - Fillet welds in scallop valleys are continuing to propagate
 - Repaired cracking at the toe of the scallop peak fillet attachments have not re-initiated in this area.
 - No cracks found in slotted and un-slotted scallop peaks

Inspection Photos of Retrofit Skirt

