

# Vibration Testing and Troubleshooting of Coke Drums

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# **OVERVIEW**

- Vibrations Basics
- Drum Dynamics
- What is Affected
- Potential Consequences
- Case study





# **VIBRATIONS BASICS**

- Response of a flexible structure to excitation (fluid flow, pressure pulsations, sloshing, etc..)
- Components
  - Mass (inertia)
  - Spring stiffness (Force–Displacement)
  - Dashpot viscous damping (Force–Velocity)
- Basic characteristics:
  - Natural frequency (or frequencies)
  - Mode shapes
- Input output
  - Force
  - Displacement







#### **DRUM DYNAMICS**









walue solution 34466. Freq = 29.547







# WHAT IS AFFECTED

- Pipes and pipe supports
- Base-plate bolts and grout
- Non-structural (stairs, lights, guardrails,..)
- Machinery (elevator, pumps, ..)
- Superstructure (concrete and steel)
- Foundation system (substructure and soil)

#### Humans!





# POTENTIAL CONSEQUENCES

- Operator discomfort or fatigue
- Poor performance
- Acceleration of corrosion damage
- Interruption of operations during repairs
- Fatigue cracks (leaks/ fires)
- Bodily injuries





# WHAT TO DO?

- Measure
  - Vibrations
  - Strain
  - Process variables
- Analyze
  - Stress / fatigue
  - Human tolerance
- Mitigate
  - Process
  - Structure





# CASE STUDY - 1

- Two-drum unit
- "Significant" vibrations in structure
- Conflicting views on
  - When they vibrate the most
  - Which one vibrates more
- Failures
  - Piping supports
  - Anchor bolts
  - Base plate grout
- Is the unit safe?
- Can the process be optimized to minimize vibrations?







### DRUM DESIGN AND OPERATION

1996 API Coke Drum Survey, 2003

- 16 feet in diameter relatively small for the mid 80's
- 76 feet in height average to tall
- Slender drums
- Very common 1 <sup>1</sup>/<sub>4</sub> Cr -1/2Mo material
- 17 hour fill cycle average to relatively slow operation



# OBJECTIVES

- Monitor vibrations in the drums, piping, and structure.
- Obtain synchronized temperature and strain measurements.
- Determine the timing and characteristics of maximum vibrations.
- Determine severity of dynamic stresses and potential for fatigue damage in the structure.
- Conduct sensitivity analysis of vibrations versus process variables.



## PROCEDURE

- Installed 33 sensors and two data acquisition systems on the drums, piping, and structure.
- Monitored the unit for a period of 20 days (14 cycles)
- Processed and analyzed the data in time and frequency domain.
- Analyzed the correlation between key process variables with vibration, strain, and temperature measurements.





# INSTRUMENTATION

Vibration data acquisition system

16 channel unit

16 seismic accelerometers @ 102 samples per second

2 strain gages (low-speed DC channels)

Temperature data acquisition system

StrainDAQ unit

16 thermocouples

1 sample per two seconds

Data collection was continuous without interruption during the entire monitoring period.







#### **Example Quench Transient**



# Integrated Displacement Data



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# **Vibration Versus Flow Rate**







# Strain Gauge Data





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### CASE-1 SUMMARY

- Drum vibration magnitude was maximum during the quench part of the cycle in the East-West direction.
- The two drums vibrated in a comparable manner both from magnitude and frequency standpoints.
- The maximum recorded peak displacements were 0.58, 0.35, and 0.23 inches for the drums, the piping, and the structure, respectively.
- Measured dynamic strains in the structure were below the fatigue-inducing levels
- The correlation between vibration levels and recorded process parameters was established





# CASE STUDY -2

- Four-drum unit.
- Blow-down line.
- Cracks and leaks.
- Angled-tee joint.
- Thermal cycles.
- 3D loads.
- Vibrations.
- Why?
- How to fix it?







# **ACTION PLAN**

- 1. Instrumentation
  - Strain gages
  - Thermocouples
- 2. Extraction of loading conditions
- 3. Finite element analysis
- 4. Fatigue Assessment





### INSTRUMENTATION

#### **Intrinsically Safe Instrumentation System**







#### FIELD MONITORING







#### **TEMPERATURE MEASUREMENTS**





#### THERMAL GRADIENTS

#### Every forth cycle





•t1

t2

t3

t4

#### CLOSEUP







## PAD THERMAL GRADIENTS







## **Extracted load cases**

#### Five thermal profiles during the coking cycle













#### **Von Mises Stress in the pipe**







#### **Von Mises Stress inside the pad**







### **Fatigue Assessment**

- Maximum thermal stress range is 92 ksi.
- Alternate stress is 49.8 ksi.
- Correction for E at 400<sup>0</sup>F
- Fatigue life is 4,441 cycles.





# **CASE-2 SUMMARY**

- Vibrations and pressure are not the main problem.
- The failure is caused by severe thermal transients that generate 92 ksi stress range in the pipe at the location of cracks.
- Recommendations to minimize stresses and increase fatigue life:
  - Redesigned integral fitting.
  - Fatigue-resistant welds.



# **Questions?**

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