

Mechanical Seals in Decoker Units Michael Huebner – Staff Engineer

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Agenda

- Decoker Unit Application Overview
- Critical Applications
- Support Systems
- Troubleshooting
- Open Discussion

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Decoker Unit Application Overview

- What makes a mechanical seal application challenging?
 - > High temperature
 - Coking
 - > High pressure
 - > Solids
 - > Poor system performance

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High Temperature Sealing

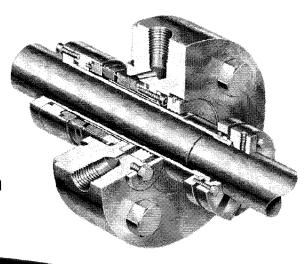
- Basic Seal Design Criteria
 - Materials must withstand maximum temperatures, typically up to 800 F (427 C)
 - Components must tolerate thermal expansion differences
 - Outboard seal must prevent atmospheric coking or solidification
 - Assembly must accommodate high axial shaft movement during warm-up

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Typical High Temperature Seal

- BXRH
- Single seal
- Welded metal bellows
- Flexible stator
- Plan 62 steam quench

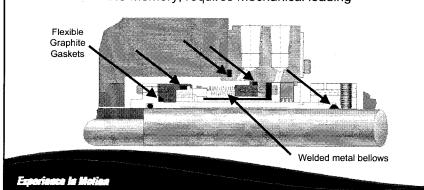


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Material Selection

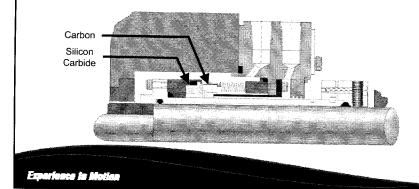
- · Welded metal bellows: Alloy 718
- Excellent HT physical properties
 - Good corrosion resistance including H₂S
- Flexible graphite gaskets
 - > Excellent HT properties and corrosion resistance
 - > No memory, requires mechanical loading





Material Selection

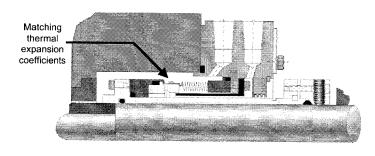
- · Silicon Carbide and Carbon seal faces
 - > Excellent HT properties and corrosion resistance
 - > Use two hard faces with abrasive media
- Alloy metals or 316 SS for all other metal components
 - > Maximize corrosion resistance
 - > Don't compromise with "cheap" bolts/screws





Tolerating Thermal Expansion

- · Match thermal expansion in shrink fit
 - Carbon or silicon carbide seal face requires a low expansion bellows flange
 - High Chrome alloy is required with high Sulfur or Hydrogen Sulfide content



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Thermal Expansion Rates

- Approximate rates of thermal expansion
 - > AM-350
- 7 E-6 in/in °F
- > Alloy C-276
- 6 E-6 in/in °F
- > Alloy 718
- 7 E-6 in/in °F
- > 316 SS
- 9.5 E-6 in/in °F
- > Alloy 42
- 3 E-6 in/in °F
- > Carbon
- 3 E-6 in/in °F

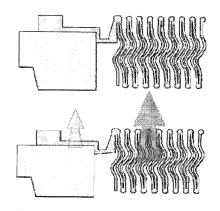
> SiC

3 E-6 in/in °F

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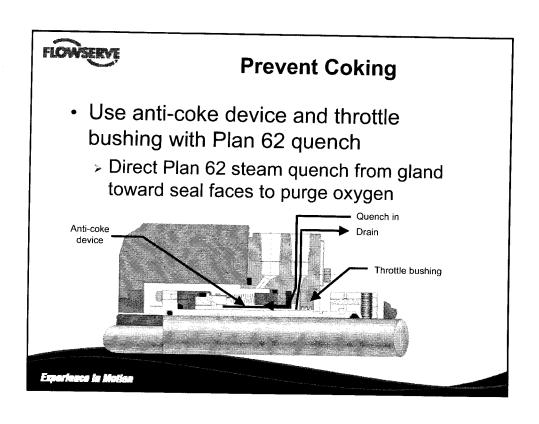


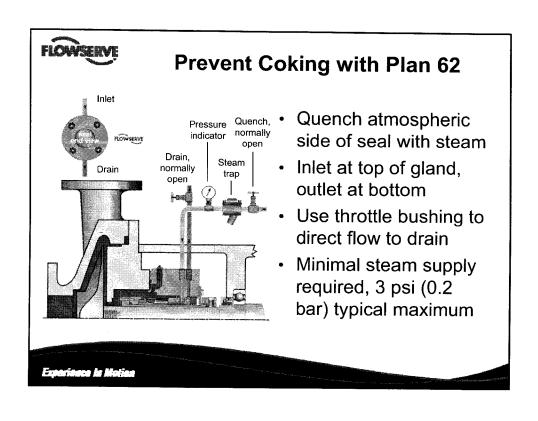
Tolerating Thermal Expansion

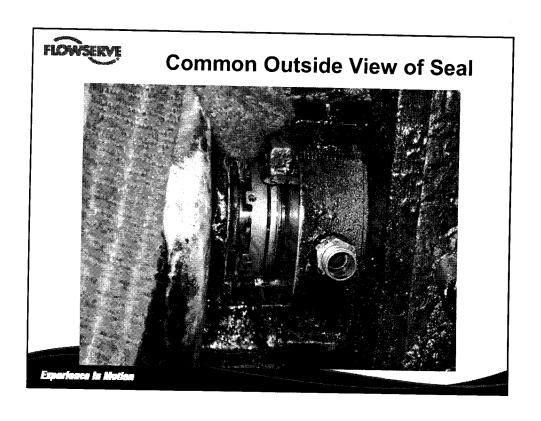


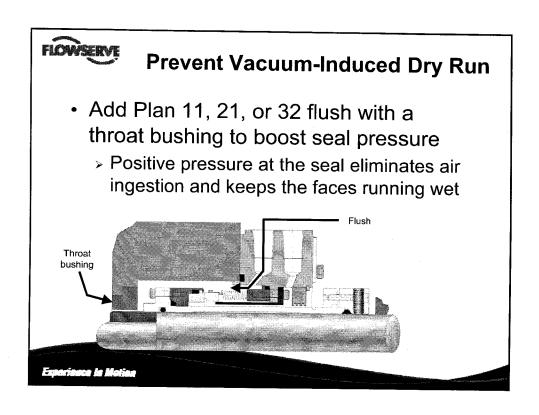
- At ambient temp., the seal face is lapped flat
- As temp. increases, the bellows diaphragms expand faster than the bellows flange & face
- Expansion is absorbed in the hinge
- · Seal face remains flat

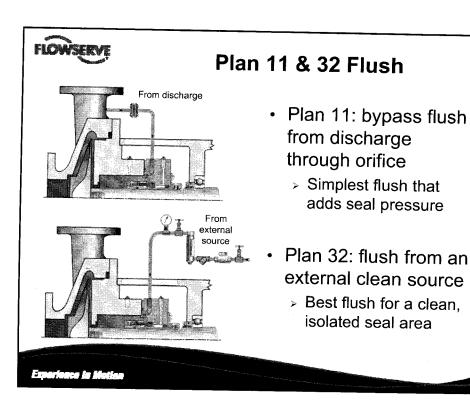
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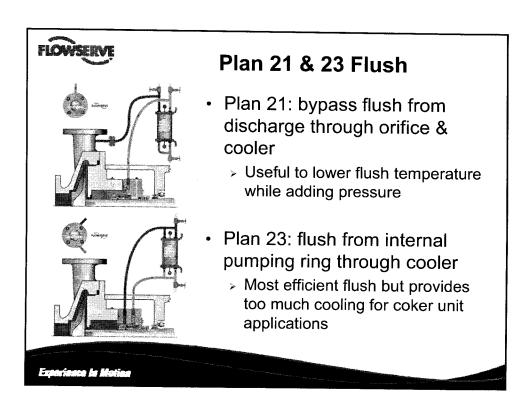








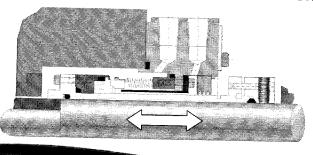






Accommodate Axial Growth

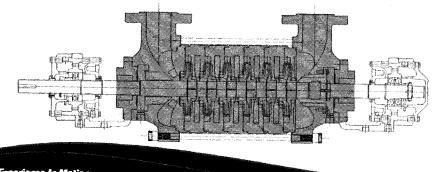
- Metal bellows allow long axial travel
 - > During pump warm-up, the shaft may have faster axial growth relative to the casing
 - > Bellows absorb transient shaft movement

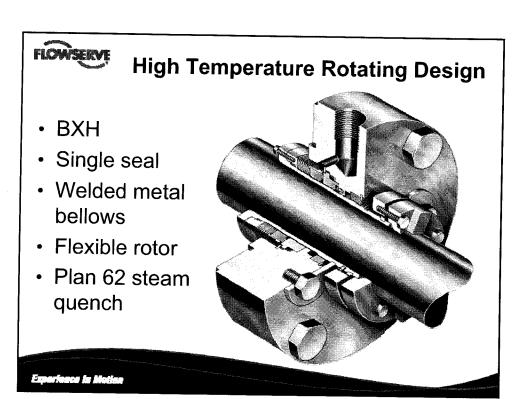




Starting Hot Pumps

- Slow warm-up is essential for the shaft and housing to grow together
 - > Rapid warm-up increases stresses on pump parts, alignment, piping, and seals







Rotating vs. Stationary Bellows

- Stationary Bellows
 - > Allows higher speeds
 - > Absorbs shaft misalignment
 - > Steam quench keeps bellows ID clean
- Rotating Bellows
 - > Self-cleaning bellows OD
 - Generally shorter, used in dual seals

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High Temperature Dual Seals

- Dual seals offer some advantages in high temperature applications
 - Eliminate coking conditions on atmospheric side
 - > Operate under vacuum conditions
 - > Add safety measure
 - > Assist monitoring requirements

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HT Dual Seal Considerations

- Wet barrier seals
 - Barrier fluid must be suitable for sustained high temperature condition
 - Reservoir must be larger than normal and cooling coils are mandatory
 - Design must allow room for barrier fluid to circulate under IB seal
 - How to rectify up to 800 F (427 C) pump with 200 F (93 C) maximum barrier fluid?

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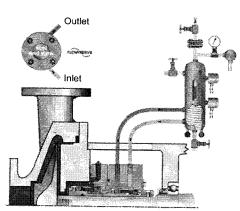
Plan 32 Flush for Dual Seals

- Flush isolates, cools, and protects inboard seal
 - > Turn on flush before starting and leave it on after stopping pump
 - Flush fluid must be compatible with process, may come from similar process
 - Flow rate is low, use throat bushing to isolate seal area
 - > Flush source must be reliable

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Dual Seal Plan 52 & 53A



- Plan 52: unpressurized, with orifice in vent
- Plan 53A: pressurized
- Safety backup
- Prevents coking
- Provides some cooling to the inboard seal
- Can protect against pressure fluctuations

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Minimize line losses Use large diameter tubing Only upward sloping lines Use long radius bends Minimize component losses Optimize for thermal siphon Check rotation direction

Test for leaks

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• Hard piping
• Fittings
• Valves
• Long runs
• Excess elevation
• Vapor traps
• Not self-venting

3 ft (1 m)

minimum

Warm

Cool



HT Dual Seal Considerations

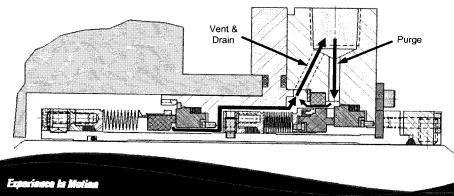
- Dry containment (outboard) seals
 - Plan 72 steam purge must flow between seals to prevent inboard seal coking
 - Dual seal must drain condensate and process leakage to Plan 75 collection vessel
 - Containment seal does not protect inboard seal from running dry under heavy vacuum
 - > Low pressure steam may leak into process
 - > Most applications still require inboard flush

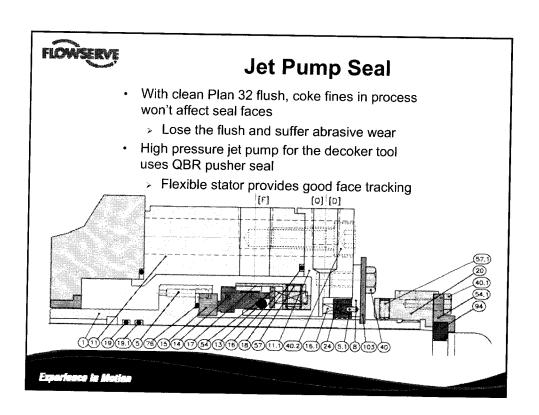
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HT Containment Seal

- Containment seals were introduced in the 2nd Edition of API 682
 - No applications in decoker units were found but could be viable alternative







Conclusions

- Decoker unit seal applications are not so different from other parts of the refinery - focus on fundamental seal practices
- High temperature coker feed/charge pump seals require careful attention to seal design and system operation
- Jet pump seals achieve long life when coke fine-induced abrasion is prevented

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